

# CH EN 3453 – HEAT TRANSFER – FALL 2014

## HOMework #2

Due Friday, September 5, at 4:00 PM

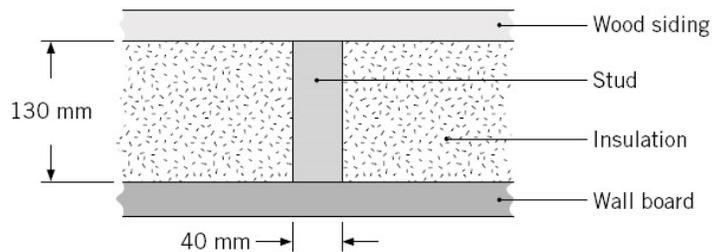
Turn in to the CH EN 3453 basket at the main desk of the Chemical Engineering offices (MEB 3290)

Help session Wednesday, September 3 at 4:30 p.m. in MEB 2325

For this assignment, you must...

- show your work
- write legibly and not in micro-font
- circle or put a box around your final answer

- (15 pts) Define the following physical properties. Include whether it is a thermodynamic or transport property as well as correct units (AE and SI) and the corresponding symbol.
  - Thermal conductivity
  - Kinematic viscosity
  - Density
  - Specific heat
  - Thermal diffusivity
- \* (9 pts) The steady-state temperature distribution in a one-dimensional wall of thermal conductivity  $50 \text{ W/m}\cdot\text{K}$  and thickness  $50 \text{ mm}$  is observed to be  $T(^{\circ}\text{C}) = a + bx^2$  where  $a = 200^{\circ}\text{C}$ ,  $b = -2000^{\circ}\text{C}/\text{m}^2$  and  $x$  is in meters.
  - What is the heat generation rate  $\dot{q}$  in the wall?
  - Determine the heat fluxes at the two wall faces. In what manner are these heat fluxes related to the heat generation rate?
- \* (9 pts) A thermopane window consists of two pieces of glass  $7 \text{ mm}$  thick that enclose an air space  $7 \text{ mm}$  thick. The window separates room air at  $20^{\circ}\text{C}$  from outside ambient air at  $-10^{\circ}\text{C}$ . The convection coefficient associated with the inner (room-side) surface is  $10 \text{ W/m}^2\cdot\text{K}$ . If the convection coefficient associated with the outer (ambient) air is  $h_o = 80 \text{ W/m}^2\cdot\text{K}$ , what is the heat loss through a window that is  $0.8 \text{ m}$  long by  $0.5 \text{ m}$  wide? Neglect radiation, and assume the air enclosed between the panes to be stagnant (i.e., ignore convection to and from the air between the two panes – but don't neglect conduction through that air).
- \* (9 pts) Consider a composite wall that includes an  $8\text{-mm}$ -thick hardwood siding,  $40\text{-mm}$  by  $130\text{-mm}$  hardwood studs on  $0.65\text{-m}$  centers with glass fiber insulation (paper faced,  $28 \text{ kg/m}^3$ ) and a  $12\text{-mm}$  layer of gypsum (vermiculite) wall board. What is the thermal resistance associated with a wall that is  $2.5 \text{ m}$  high by  $6.5 \text{ m}$  wide (having 10 studs, each  $2.5 \text{ m}$  high)?



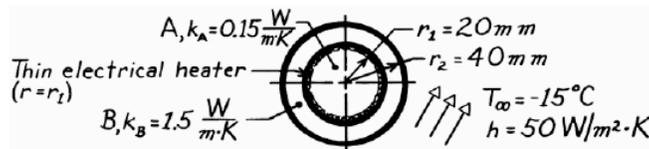
More problems on the other side...

5.\* (9 pts) A 304 stainless steel tube used to transport a chilled pharmaceutical has an inner diameter of 36 mm and a wall thickness of 2 mm. The pharmaceutical and ambient air are at temperatures of 6°C and 23°C, respectively, while the corresponding inner and outer convection coefficients are 400 W/m<sup>2</sup>·K and 6 W/m<sup>2</sup>·K, respectively.

- (a) What is the heat gain per unit tube length?
- (b) What is the heat gain per unit length if a 10-mm-thick layer of calcium silicate insulation ( $k_{\text{ins}} = 0.050 \text{ W/m}\cdot\text{K}$ ) is applied to the outside of the tube?

6.\* (9 pts) A thin electrical heater is inserted between a long circular rod and a concentric tube with inner and outer radii of 20 and 40 mm. The rod (A) has a thermal conductivity of  $k_A = 0.15 \text{ W/m}\cdot\text{K}$ , while the tube (B) has a thermal conductivity  $k_B = 1.5 \text{ W/m}\cdot\text{K}$  and its outer surface is subjected to convection with a fluid of temperature  $T_\infty = -15^\circ\text{C}$  and heat transfer coefficient 50 W/m<sup>2</sup>·K. The thermal contact resistance between the cylinder surfaces and the heater is negligible.

- (a) Determine the electrical power per unit length of the cylinders (W/m) that is required to maintain the outer surface of cylinder B at 5°C.
- (b) What is the temperature at the center of cylinder A?



7. (20 pts) A furnace wall is made up of a 0.25-m thick layer of fireclay brick ( $k = 2.5 \text{ W/m}\cdot\text{K}$ ), a 0.20-m thick layer of insulating cement ( $k = 0.16 \text{ W/m}\cdot\text{K}$ ) and a 0.10-m outer layer of masonry face brick. Furnace gases on the inside of the furnace are at 1000°C with a convective heat transfer coefficient of 115 W/m<sup>2</sup>·K while the outer wall is exposed to air at 20°C ( $h = 25 \text{ W/m}^2\cdot\text{K}$ ).

- (a) Ignoring radiation, compute heat loss per square meter of wall.
- (b) Determine the temperature of the outer surface of the furnace.
- (c) If the maximum temperature of the masonry face brick cannot exceed 50°C, by how much must the insulating cement thickness be adjusted to satisfy this requirement?

8. (20 pts) The case covering the back of my cell phone is soft vulcanized rubber 1/8 inch thick. On the front I have a screen protector of 1/16-inch thick soda lime glass. The phone itself is 2.3 inches wide, 4.5 inches tall and 0.4 inches thick. Each Friday night I watch Star Wars on my phone, which results in heat production of 10 Watts. I notice that the front of the phone (between the phone and glass) is 3°C cooler than the back of the phone (between the phone and rubber). If the air temperature on my windy back porch is 28°C and the convective heat transfer coefficient is 150 W/m<sup>2</sup>·K, what are the temperatures at (a) the phone-rubber interface and (b) the phone-glass interface? Ignore any heat loss through the edges of the phone (i.e., consider only heat loss through the front and back).